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(71) Applicant: Hewlett-Packard Company
Mail Stop 20 B-O 3000 Hanover Street
Palo Alto California 94304(US)

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(72) Inventor: Stephens, Thomas P.
420 Ipswich Road
Boxford Massachusetts 01921(US)

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(74) Representative: Schulte, Knud, Dipl.-Ing.
c/o Hewlett-Packard GmbH Europ. Patent- und
Lizenzzabteilung Postfach 1430 Herrenberger Strasse 130
D-7030 Böblingen(DE)

(54) Non-loosening leur nut.

(57) An assembly for axially coupling male and female tubes including a nut having internal threads on a polygonally shaped inner surface at one end that can be screwed onto a projection on one of the tubes so as to draw a collar at the other end of the nut into axial contact with a shoulder on the other end of said tubes so as to draw the tubes together, the said projection being such as to ride in said threads at points along the surface of at least one side of said polygon so as to bend it outwardly from its unstressed position.

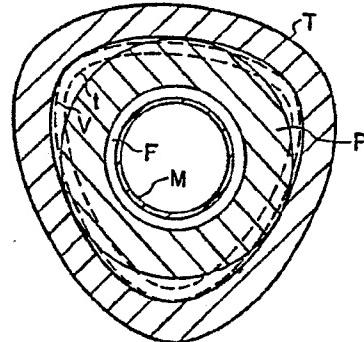


FIG 2

EP 0 204 311 A2

DESCRIPTION
NON-LOOSENING LUER NUT

This invention relates to improvements in an assembly employing a hollow Luer nut to draw the ends of two hollow tubes together so as to form a continuous passageway therethrough. In this assembly, the outer end of a male tube becomes smaller towards an end hereof so as to form a conical surface that can be seated in an axial conical recess in an end of the female tube so as to form a sealed joint. An outer annular ridge is formed adjacent to the larger end of the outer conical surface of the male tube, and an outer annular projection is formed about the end of the female tube. The Luer nut has an annular collar at one end and internal threads in the other, and does not deform during use.

It is positioned with the male tube passing through the collar, the annular ridge on the male tube inside the collar, and the annular projection on the female tube engaging the threads. As the nut is screwed onto the annular projection, the collar contacts the ridge on the male tube and moves it towards the female tube until the conical end of the male tube is seated in the conical recess of the female tube so as to form a seal.

The normal Luer nut has a 10-pitch thread with 2 starts. Less than one turn of the nut is required to couple or decouple the tubes.

Unfortunately, however, the nut is often loosened during normal handling or just relaxing and the seal broken. This is due to the high angular contact of the threads. The thread contact point is a steep wedge that is likely to unloosen because of the steep angle of contact. Some improvement was attempted by reducing the diameter of the threads as the interior of the nut was approached so as to increase the frictional torque as the nut was screwed into position. In some assemblies, however, the dimension of the threads in the nut and of the annular projection at the end of the female tube, although within normal manufacturing tolerances, were such that the frictional torque became too great for an operator to easily overcome before the male tube seated firmly enough in the female tube to form a seal. This is prevalent in a round female flange and is caused by the round shape of the nut being extended radially outward until the nut resists further radial expansion because the walls of the nut are in tension or hoop stress.

In accordance with the present invention, the internal cross-section of the threaded section of the Luer nut is generally in the shape of a polygon that is deformable rather than stiff as in the prior art. In an axial view, the shape of the projection at the end of the female tube may be annular as in previous assemblies or it may have other shapes such as a rectangle. In any case, a portion of the periphery of the projection contacts one or more sides of the polygon. Wherever contact is made, the radial dimensions of the projection are such that the side is bent radially outward so that it acts like a spring and exerts a radially inward force on the projection. The resulting force of friction creates a frictional torque which does not prevent the operator from turning the nut, but which is sufficient to prevent the nut from becoming loose during normal use.

If the male tube is nearly seated in the female tube when the projection of the female tube reaches a point where the nut can be received onto it, male tube and female tube will be held very nearly in axial alignment so that the periphery of the projection may contact a side of the polygon at only one point. Otherwise it would be necessary for the periphery to make contact with a number of sides of the polygon so that the female tube is held in axial alignment with the nut. The latter structure is preferable in our case, however.

Other features and advantages can be recognized from the drawings and the description thereto.

Figure 1 shows what would be seen in an axial plane of a Luer nut assembly constructed in accordance with this invention at a point just before the male tube is seated in the female tube;

Figure 2 is an axial view of an assembly incorporating the invention in which the interior cross-section of the threaded portion of the Luer nut is generally triangular and the periphery of the projection at the end of the female tube is annular;

Figure 3 is an axial view of an assembly incorporating the invention in which the interior cross-section of the threaded portion of the Luer nut is generally triangular and the periphery of the projection at end of the female tube has two opposed arcs; and

Figure 4 is an axial view of an assembly incorporating the invention in which the interior cross-section of the threaded portion of

the Luer nut is generally triangular and the periphery of the projection at the end of the female tube is approaching a square.

Figure 1 illustrates what would be in an axial plane of an assembly incorporating the invention. A hollow male tube M as an exterior conical surface CS and an annular ridge R adjacent the larger end of the surface CS. A hollow female tube F has an axial conical recess S into which the surface CS can be made to fit so as to form a seal and an outwardly extending projection P. A hollow Luer nut N is shown having a collar C at one end thereof and an internal thread t in the other end E that engages the projection P. As the nut N is rotated in one direction about its axis with respect to the female tube F, the projection P advances in the threads t and the female tube F is moved toward the male tube M. Finally, the conical surface CS and the conical recess S are seated as to form a seal.

Reference is now made to Figure 2 which illustrates what is seen in an assembly in a plane that is perpendicular to the axis of the female tube F and which passes through the projection P. Inasmuch as Figure 1 is in itself a section, Figure 2 is not a section of Figure 1 but would be located at VV therein. In Figure 2, the internal cross-section of the end E of the nut N is a polygon generally in the form of an equilateral triangle T. The projection P is an annulus which contacts the midpoints of the inner sides of the triangle so as to bend the sides outward. The undistorted shape of the inner surface of the triangle T is shown by a dash-dot line V. Although not in the plane shown in Figure 2, the location of the threads t, when the sides of the triangle T are bent outward by the projection P, is shown by a dashed line. Where the periphery of the annular projection P contacts the sides of the triangle, it engages the threads t so that the nut N can be screwed onto the threads. The fact that the projection P is not engaged with the threads t at every point does not prevent this action from occurring.

An important aspect of this invention is the fact that wherever the periphery of the projection P on the female tube F contacts a side of the polygon formed inside the threaded end E of the nut N, its radial dimension is such as to bend the side outward. The side acts as a spring and creates a frictional torque sufficient to prevent the nut from being loosened during use.

In Figure 2, the polygon formed by the internal cross-section of the end E of the nut N is an equilateral triangle T with rounded corners. Sharp corners could be used if desired. The periphery of the projection P is annular and is of such radius as to contact the central points of the sides of the triangle T and bend the sides outward from their unstressed position shown by the dashed line V. The bending can also be seen in Figure 1 where the projection P contacts the inside of the threaded end E of the nut N.

In Figure 3, the polygon formed by the internal cross-section of the end E of the nut N is an equilateral triangle T as in Figure 2, but the periphery of the projection P' is formed by two arcs A₁ and A₂ on opposite sides of the axis of the female tube F and having that axis as a center, and by two parallel lines on either side of the axis and intersecting the arcs at corners C₁, C₂, C₃ and C₄. The central portion of the arc A₁ contacts the center of the top side of the triangle T, and the corners C₃ and C₄ respectively contact the other sides at points below the center. The sides are bent out at the points of contact so as to create the frictional torque required. The dash-dot line V is not shown.

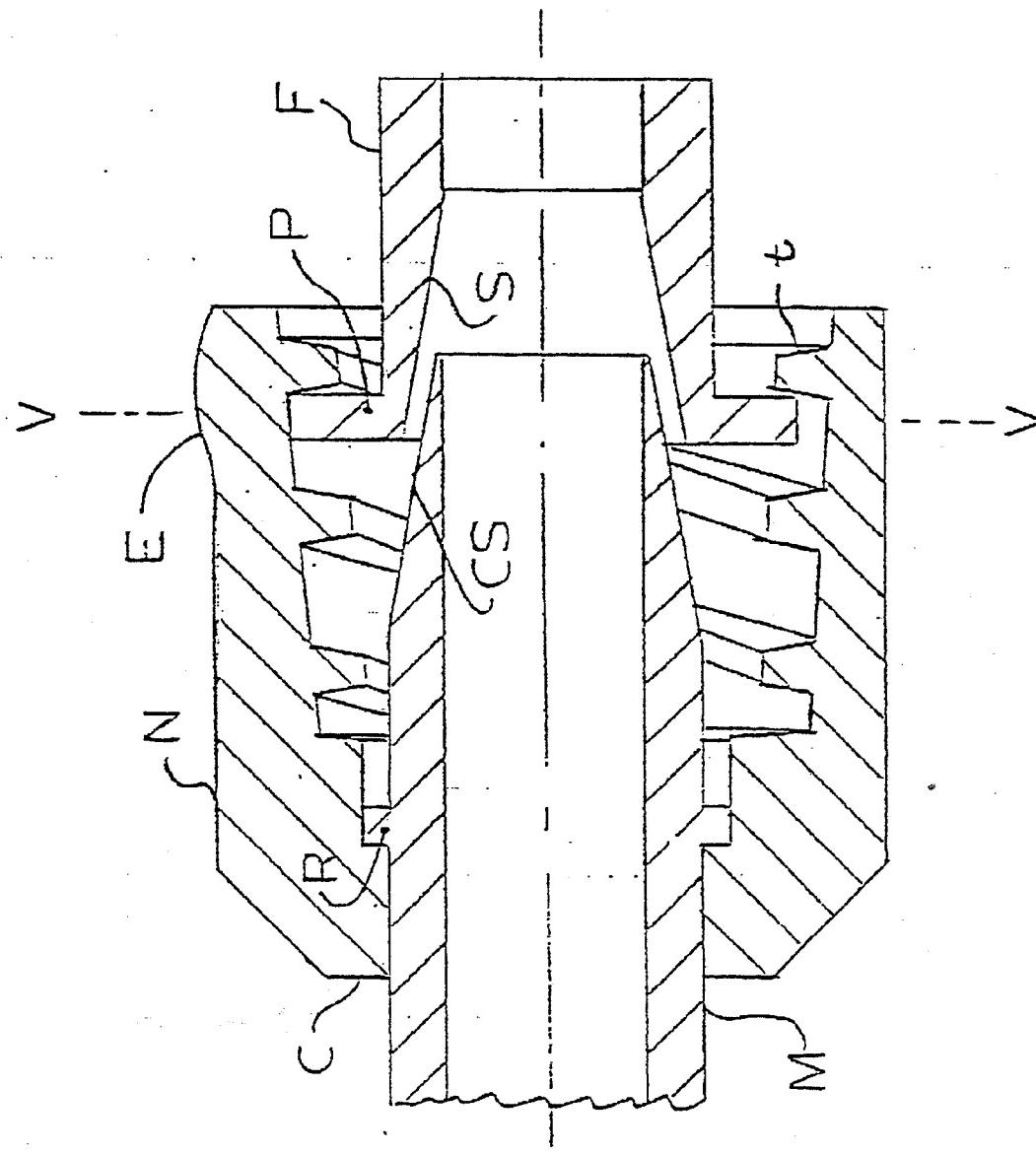
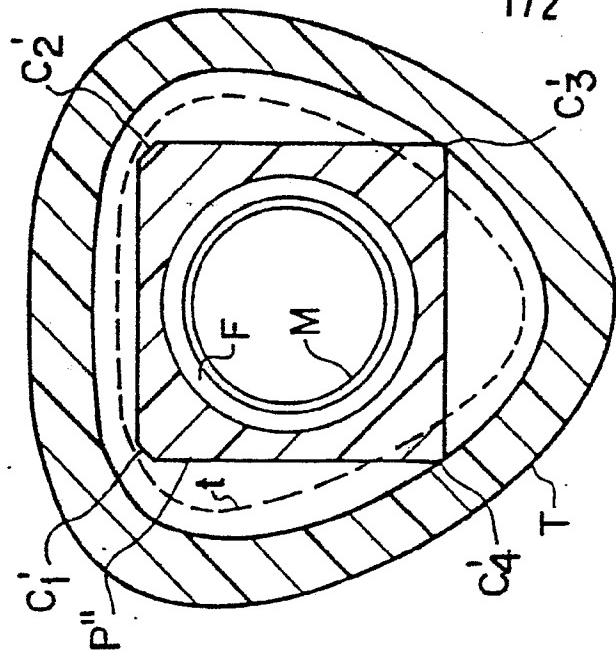
Figure 4 is the same as Figure 3 except that the periphery of the projection P'' is in the form of a square that contacts the triangular polygon T at its lower corners C'3 and C'4. The fact that the upper corners C'1 and C'2 do not initially contact the polygon presents no problem because the male and female tubes will stay substantially in alignment when they are nearly seated. Note, however, that the top side of the triangle T is not bent outward.

It is apparent that the outer surface of the end E of the nut N can have a cross-section that is shaped differently from that of its inner surface.

In order to increase the frictional torque as the nut is being screwed onto the external projection of the female tube, the radial dimensions of the polygon formed by the inner surface on which the threads are formed can be decreased as the interior of the nut is approached by an amount that is greater than that required to extricate the mold, but this should not be such as to cause binding.

CLAIMS

1. An assembly comprising
 - a female hollow tube,
 - a male hollow tube and
 - a hollow nut,characterized in that
 - (a) said female tube (F) has
 - an exterior projection (P) at the end, said projection(P) having a given periphery in a plane perpendicular to the axis of the tube,
 - means defining a conical recess (S) in one end of said female tube (F),
 - (b) said male tube (M) has
 - an external surface (CS) at one end thereof that is conical in shape so as to fit into said conical recess (S) when the ends of said female and male tubes are axially drawn together,
 - (c) a shoulder (R) is formed on the exterior of said male tube (M) at a point adjacent the larger end of said exterior conical surface (CS),
 - (d) said hollow nut (N) has threads (t) formed on an interior surface about an axis of said nut (N), said interior surface having a cross-section in a plane perpendicular to said axis that is in the form of a polygon,
 - (e) said hollow nut (N) has an inwardly extending collar (C) at the other end through which said male tube (M) passes, said collar (C) being more remote from the interior of said nut (N) than said shoulder (R),
 - (f) the dimensions and shape of the periphery of said projection (P) are such that it contacts the interior surface of said nut (N) at least one point.
2. An assembly according to Claim 1 wherein the periphery of said exterior projection (P) is annular.
3. An assembly according to Claim 1 wherein the periphery of said exterior projection (P) is rectangular.
4. An assembly according to Claim 1 wherein the periphery of said exterior projection (P) is in the form of parallel lines connected at opposite ends by curved lines.



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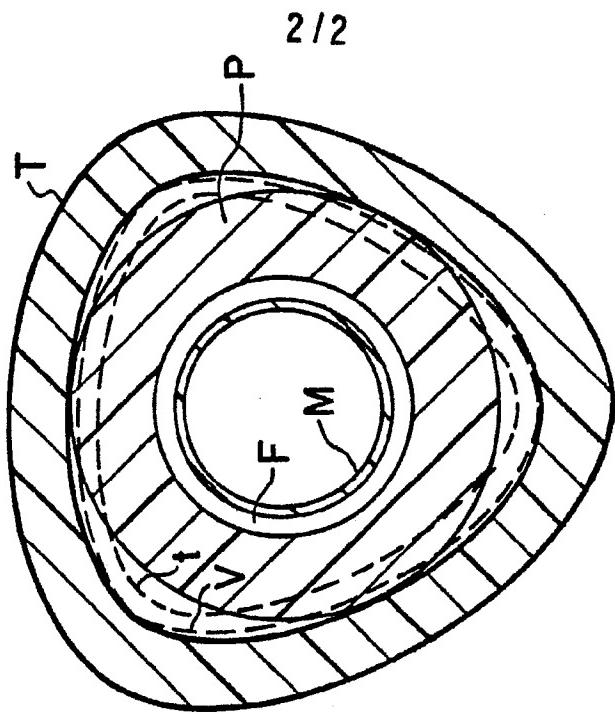


FIG 2

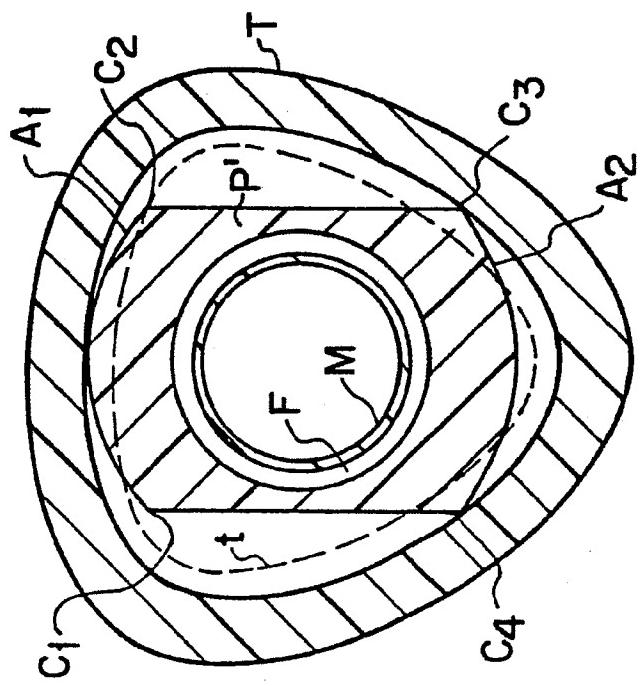


FIG 3



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(71) Applicant: Hewlett-Packard Company
Mail Stop 20 B-O 3000 Hanover Street
Palo Alto California 94304(US)

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(72) Inventor: Stephens, Thomas P.
420 Ipswich Road
Boxford Massachusetts 01921(US)

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(74) Representative: Schulte, Knud, Dipl.-Ing.
c/o Hewlett-Packard GmbH Europ. Patent- und
Lizenzzabteilung Postfach 1430 Herrenberger Strasse 130
D-7030 Böblingen(DE)

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(54) Non-loosening leur nut.

(57) An assembly for axially coupling male (M) and female (F) tubes including a nut (N) having internal threads on a polygonally shaped inner surface (T) at one end (E) that can be screwed onto a projection (P) on one of the tubes (M)(F) so as to draw a collar (C) at the other end of the nut (N) into axial contact with a shoulder on the other end of said tubes so as to draw the tubes (M)(F) together, the said projection (P) being such as to ride in said threads at points along the surface of at least one side of said polygon (T) so as to bend it outwardly from its unstressed position.

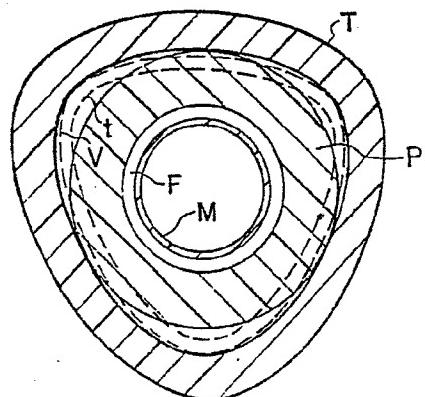


FIG 2

EP 0 204 311 A3



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EUROPEAN SEARCH REPORT

Application number

EP 86 10 7510

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.4)
Y	GB-A-2 089 921 (BAXTER TRAVENOL LABORATORIES INC.) * Whole document *	1	F 16 L 19/02 F 16 B 39/284 A 61 M 39/00
A	---	3,4	
Y	DE-A-2 557 130 (IVH INDUSTRIEVERTRETUNGEN UND HANDEL GmbH) * Figures 1,3 *	1	
A	---	1	
A	GB-A-2 146 405 (SMITH AND NEPHEW ASSOCIATED CO.) * Page 2, lines 11-56; figure 1 *	1	
A	---	1,3	TECHNICAL FIELDS SEARCHED (Int. Cl.4)
A	CH-A- 248 339 (SOCIETE D'ECLAIRAGE, CHAUFFAGE & FORCE MOTRICE) * Figure 2 *	1,2	F 16 L F 16 B A 61 M
A	BE-A- 535 526 (GAUTHIER) * Figure *	1	
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The present search report has been drawn up for all claims			
Place of search	Date of completion of the search	Examiner	
THE HAGUE	16-04-1987	SARRE	
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EUROPEAN SEARCH REPORT

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DOCUMENTS CONSIDERED TO BE RELEVANT			Page 2
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl.4)
A	GB-A-1 397 493 (USUI KOKUSAI SANGYO K.K.) * Figures 1,3 *	1	

			TECHNICAL FIELDS SEARCHED (Int. Cl.4)
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THE HAGUE	16-04-1987	SARRE	
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